

Streams of Interest

Stream	Length (km)	Width (m)	Gradient (m/m)	Aspect
Canada (F1)	1.5	21.4	0.03	South
Lost Seal (F3)	2.2	4	0.02	South
Von Guerard (F6)	4.9	3-4	0.08	North
Green (F9)	1.2	1-4	0.02	East
Delta (F10)	11.2	10	0.03	North

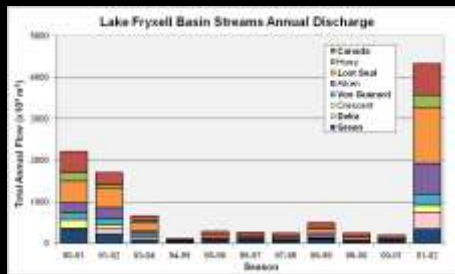
Variance in Flow

- Daily & Seasonal Variance
- Solar position and glacier face melt
- Incoming shortwave radiation absorption
- Stream Temps:
 - Range: 0 to 25 °C
 - 10°C warmer than air temperature
 - 20 °C variance in one day



Gardner, Chris. Green Creek in Lake Fryxell Basin, McMurdo Dry Valleys Long-Term Ecological Research. 23 July 2008. <http://mcmler.org/photoDB/green_creek1.jpg>

Period of Interest



Jaros, Chris. Bar Chart of Total Annual Inflow from Lake Fryxell Basin Streams. From Thesis (2002).

- One week from each season
- Simultaneous flow

Stream Temperature Regime

Definition: Range and Timing

- Driven by solar radiation
- Thermal conductivity is constant
- Compare peaks of temp regime to flow regime

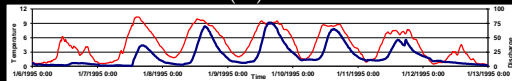


Gardner, Chris. Green Creek in Lake Fryxell Basin, McMurdo Dry Valleys Long-Term Ecological Research. 23 July 2008. <http://mcmler.org/photoDB/green_creek1.jpg>

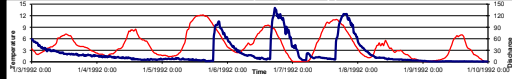
Comparison of Temp and Discharge

– Temp (°C) – Discharge (L/s)

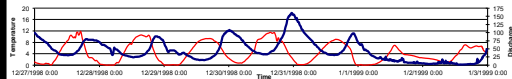
Canada Stream (F1) – Low Flow Season



Von Guerard Stream (F6) – High Flow Season

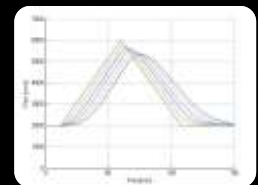


Lost Seal Stream (F3) – Moderate Flow Season



Kinematic Wave Model

- Routes pulse of variable flow
- Function of time & space
- Stream characteristics
- Physical processes
- Boundary Condition: $Q(0,t)$
- Initial condition: $Q(x,0)$



Kinematic Wave Model

Assumptions Made:

- Neglect acceleration and pressure
- Omit lateral inflow, eddy loss, & wind shear
- Steady and uniform within dx
- One-dimensional flow
- Incompressible fluid
- Width less variable than depth
- Uniform width and gradient
- Prismatic channel

St. Venant Equations

Continuity Equation:

$$\frac{d}{dt} \iiint_{c.v.} \rho dV + \iint_{c.s.} \rho \mathbf{V} \cdot d\mathbf{A} = 0 \rightarrow \frac{\partial Q}{\partial x} + \frac{\partial A_c}{\partial t} = 0$$

Momentum Equation:

$$\frac{d}{dt} \iiint_{c.v.} \mathbf{V} \rho dV + \iint_{c.s.} \mathbf{V} \rho \mathbf{V} \cdot d\mathbf{A} = \sum \mathbf{F}$$

$$\rightarrow \underbrace{\frac{1}{A_c} \frac{\partial Q}{\partial x}}_{\text{Local Acceleration}} + \underbrace{\frac{1}{A_c} \frac{\partial}{\partial x} \left(\frac{Q^2}{A_c} \right)}_{\text{Convective Acceleration}} + g \underbrace{\frac{\partial y}{\partial x}}_{\text{Pressure Force}} - g \underbrace{(S_o - S_f)}_{\text{Gravity Force Friction Force}} = 0$$

For Kinematic Wave

$$\frac{\partial Q}{\partial x} + \frac{\partial A_c}{\partial t} = 0 \quad \text{and} \quad S_o = S_f$$

Manning's Equation with $Q = UA_c$ and $R = A_c/P$:

$$Q = \frac{1}{n} \frac{A_c^{5/3}}{P^{2/3}} S_o^{1/2}$$

Solve for $A_c = \alpha Q^\beta$ with $\beta = 3/5$ and $\alpha = \left(\frac{nB^{2/3}}{\sqrt{S_o}} \right)^{3/5}$

Implicit Finite-Difference Numerical Solution

Differentiate $A_c = \alpha Q^\beta$ with respect to time:

$$\frac{\partial A_c}{\partial t} = \alpha \beta Q^{\beta-1} \frac{\partial Q}{\partial t}$$

Substitute into: $\frac{\partial Q}{\partial x} + \frac{\partial A_c}{\partial t} = 0$

$$\frac{\partial Q}{\partial x} + \alpha \beta Q^{\beta-1} \frac{\partial Q}{\partial t} = 0$$

Implicit Finite-Difference Numerical Solution

Value of Q used in $\alpha \beta Q^{\beta-1}$ is found by averaging values:

$$Q \approx \frac{Q_i^j + Q_{i+1}^{j+1}}{2}$$

The finite difference form of the linear kinematic wave is:

$$\frac{Q_{i+1}^{j+1} - Q_i^{j+1}}{\Delta x} + \alpha \beta \left(\frac{Q_i^j + Q_{i+1}^{j+1}}{2} \right)^{\beta-1} \left(\frac{Q_{i+1}^{j+1} - Q_i^j}{\Delta t} \right) = 0$$

Solved for the unknown, this equation becomes:

$$Q_{i+1}^{j+1} = \frac{\left[\frac{\Delta t}{\Delta x} Q_i^{j+1} + \alpha \beta Q_i^j \left(\frac{Q_i^j + Q_{i+1}^{j+1}}{2} \right)^{\beta-1} \right]}{\left[\frac{\Delta t}{\Delta x} + \alpha \beta \left(\frac{Q_i^j + Q_{i+1}^{j+1}}{2} \right)^{\beta-1} \right]}$$

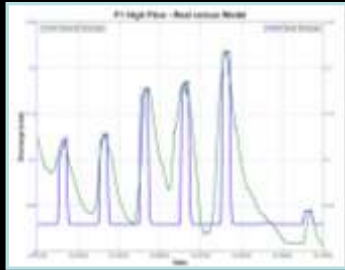
Modeling

- Use of MATLAB
- Lack of observational boundary/initial conditions
- Route water with no losses
- Sine curve provides initial input function
- Timing of direct sun on source
- Position of stream gauge
- Compare model to observed data

Modeling Results

- Captures most wave properties
- Loss of water reflects input function assumption

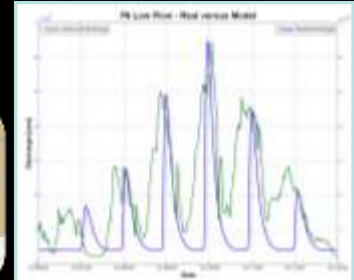
Canada Stream (F1)



Modeling Results

- Evaporation and other losses affect rising limb
- Drainage affects declining limb

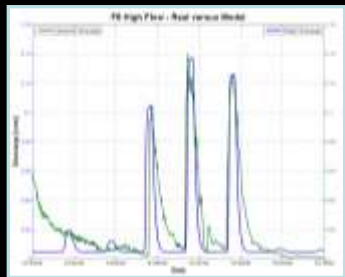
Von Guerard Stream (F6)



Modeling Results

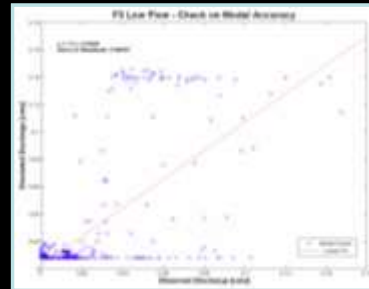
- Works best in longer stream during high flow

Von Guerard Stream (F6)



Model Accuracy

- Plot model output versus observed data
- Desire slope = 1 and small norm of residuals



Von Guerard Stream (F6):
Perfect slope,
high residuals

Future Work

- Observed data for boundary/initial condition
- Test other input function shapes
- Use glacial melt model input
- Add in evaporative and seepage losses
- Add in drainage gains during low flow
- Route flow backwards from gauge
- See what input function *should* be

Conclusions

Kinematic wave does capture the essential features of diel variation inflow from Lake Fryxell streams

e.g. Kinematic wave correctly models steep rising limb in longer streams at high flow (Von Guerard)

Acknowledgements

- NSF and CU – Boulder Environmental Engineering
- Professor Diane McKnight
- Professor Roseanna Neupauer
- Josh Koch and Karen Cozzetto
- Audrey Norvell
- Corey Wilson
- Melissa Merrill
- Chris Gardner and Matt Hoffman

Questions?

Derivation of Continuity Eq.

RTT for continuity for unsteady variable-density flow through a control volume:

$$0 = \frac{d}{dt} \iiint_{c.v.} \rho dV + \iint_{c.s.} \rho \mathbf{V} \cdot d\mathbf{A}$$

Mass Inflow: $\iint_{inlet} \rho \mathbf{V} \cdot d\mathbf{A} = -\rho(Q + qdx)$ Rate of Change of Mass Stored in CV: $\frac{d}{dt} \iiint_{c.v.} \rho dV = \frac{\partial(\rho A dx)}{\partial t}$

Mass Outflow: $\iint_{outlet} \rho \mathbf{V} \cdot d\mathbf{A} = \rho(Q + \frac{\partial Q}{\partial x} dx)$

Use substitution, assume constant fluid density and divide by qdx :

$$\frac{\partial(\rho A dx)}{\partial t} - \rho(Q + qdx) + \rho(Q + \frac{\partial Q}{\partial x} dx)$$

$$\frac{\partial A}{\partial t} - \frac{Q}{dx} - q + \frac{Q}{dx} + \frac{\partial Q}{\partial x} = 0 \quad \leftarrow \text{Take no lateral inflow, } q$$

$$\frac{\partial Q}{\partial x} + \frac{\partial A}{\partial t} = 0 \quad \leftarrow \text{Conservative form of St. Venant Continuity Equation}$$

Derivation of Momentum Eq.

RTT for continuity for unsteady nonuniform flow:

$$\sum \mathbf{F} = \frac{d}{dt} \iiint_{c.v.} \mathbf{V} \rho dV + \iint_{c.s.} \mathbf{V} \rho \mathbf{V} \cdot d\mathbf{A}$$

Sum of Forces: $\sum \mathbf{F} = F_{gravity} + F_{friction} + F_{eddyloss} + F_{windhear} + F_{pressure}$

$$\sum \mathbf{F} = \rho g A_s dx - \rho g A_f dx - \rho g A_e dx - W_f B \rho g dx - \rho g A \frac{\partial y}{\partial x} dx$$

Momentum Inflow Rate: $\iint_{inlet} \mathbf{V} \rho \mathbf{V} \cdot d\mathbf{A} = -\rho(\beta VQ + \beta v_x q dx)$ $\beta = \frac{1}{V^2 A} \iint v^2 dA$

Momentum Outflow Rate: $\iint_{outlet} \mathbf{V} \rho \mathbf{V} \cdot d\mathbf{A} = \rho \left[\beta VQ + \frac{\partial(\beta VQ)}{\partial x} dx \right]$

Net Outflow of Momentum Across CS: $\iint_{c.s.} \mathbf{V} \rho \mathbf{V} \cdot d\mathbf{A} = -\rho \left[\beta v_x q + \frac{\partial(\beta VQ)}{\partial x} dx \right]$

Rate of change of momentum stored in the control volume: $\frac{d}{dt} \iiint_{c.v.} \mathbf{V} \rho dV = \rho \frac{\partial Q}{\partial t} dx$

Derivation of Momentum Eq.

Substitute into RTT for Momentum:

$$\sum \mathbf{F} = \frac{d}{dt} \iiint_{c.v.} \mathbf{V} \rho dV + \iint_{c.s.} \mathbf{V} \rho \mathbf{V} \cdot d\mathbf{A}$$

$$\rho g A_s dx - \rho g A_f dx - \rho g A_e dx - W_f B \rho g dx - \rho g A \frac{\partial y}{\partial x} dx = -\rho \left[\beta v_x q + \frac{\partial(\beta VQ)}{\partial x} dx \right] + \rho \frac{\partial Q}{\partial t} dx$$

Assume constant density & divide through by ρdx , and replace V with Q/A_c :

$$\frac{\partial Q}{\partial t} + \frac{\partial(\beta Q^2/A)}{\partial x} + gA \left(\frac{\partial y}{\partial x} - S_o + S_f + S_c \right) - \beta q v_x + W_f B = 0$$

Neglect lateral inflow, wind shear, and eddy loss. Assume $\beta=1$. Use kinematic wave: neglect acceleration and pressure terms:

$$g(S_o - S_f) = 0 \rightarrow S_o = S_f$$

Celerity and Courant Condition

Wave: A variation in flow

Celerity: Velocity with which variation travels downstream

Celerity is different than water velocity

Kinematic wave celerity can be described as:

$$c_k = \frac{dx}{dt} = \frac{dQ}{dA} = \frac{1}{B} \frac{dQ}{dY} \quad \text{where } y \text{ is depth and } dA = B dy$$

Can be solved as: $c_k = \frac{Q^{1-\beta}}{\alpha \beta}$ where $\alpha = \left(\frac{nB^{2/3}}{\sqrt{S_o}} \right)^{3/5}$ and $\beta = 3/5$

Courant Condition: Necessary but insufficient for stability

$$\Delta t \leq \frac{\Delta x_i}{c_k}$$

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For Kinematic Wave

$$\frac{\partial Q}{\partial x} + \frac{\partial A_c}{\partial t} = 0 \quad \text{and} \quad S_o = S_f$$

Manning's Equation: $U = \frac{C_o}{n} R^{2/3} S_o^{1/2}$

$S_e \approx S_o$, $C_o=1$, and $R = A/P$: $U = \frac{1}{n} \left(\frac{A_c}{P} \right)^{2/3} S_o^{1/2}$

With $Q = UA$: $Q = A_c \frac{1}{n} \left(\frac{A_c}{P} \right)^{2/3} S_o^{1/2} \rightarrow Q = \frac{1}{n} \frac{A_c^{5/3}}{P^{2/3}} S_o^{1/2}$

Solve for $A_c = \alpha Q^\beta$ with $\beta = 3/5$ and $\alpha = \left(\frac{nP^{2/3}}{\sqrt{S_o}} \right)^{3/5} = \left(\frac{nB^{2/3}}{\sqrt{S_o}} \right)^{3/5}$

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